IN THE SPECIFICATION:

Please change paragraph 0010 to read:

[0010] By contrast, in the present invention the axial length of a turbine blade is at least 60%, and preferably more than 65%, of the radial extent of the respective turbine blade. An axial blade length which is approximately 70% of the radial extent of the blade has been found to be expedient and easily controllable. If possible, a value of no more than approximately 80%, but at most, approximately 100%, should not be exceeded for the axial length of the blades relative to their radial extent.

Please change paragraph 0012 to read:

[0012] In principle, for the turbine blades the front face has a lesser radius of curvature, at least along a portion thereof, than the corresponding opposite portion of the back face. However, the radially outer portion of each turbine blade also has a lesser radius of curvature, on both the outer face and the inner face, than the corresponding inner portion of these faces. Even where, for example, the outer portion has a lesser radius of curvature on the front face than the outer portion of the back face, the radially inward portion of the front face nevertheless has a greater radius of curvature than the radially outer portion of the back face.

Please change paragraph 0013 to read:

[0013] In a preferred embodiment of the invention, the radii of curvature of the inward portions of the front face and of the back face are relatively ~~close together~~ similar. The radius of curvature of the back face is preferably between 0 and 10% greater, in the radially inward portion, than the radius of curvature of the inward portion of the front face. The radius of curvature of the inward portion of the back face may also, if necessary, be up to 5% less than the

radius of curvature of the radially inward portion of the front face or, conversely, it may also be up to 15% greater.

Please change paragraph 0022 to read:

[0022] Fig. 1 shows a turbine wheel with carrier plate 1, in the form of a circular disk, which has an external diameter "D." On the outer margin of carrier plate 1 is a ring of turbine blades 2, the outer edges of the turbine blades being located approximately on the outer circumference, whereas the inner edges of the blade wheel ring define an inner diameter "d," as also indicated in Fig. 2. Specifically, the values for "d" are between about 40 and 48 mm, preferably about 44 mm, and the values for "D" are between about 50 and 60 mm, preferably ~~27.5 mm~~ 55 mm, the difference "D-d" being approximately 20% of "D." The corresponding radii, of course, are half the respective diameter values.

Please change paragraph 0029 to read:

[0029] Overall, for the four radii of curvature of the front and back faces in the preferred version of the invention, the relationship: $R_4 < R_2 < R_3 < R_1$ applies. The factor between R_2 and R_4 is approximately 1.3, the factor between R_3 and R_2 is approximately 2, and the factor between R_1 and R_3 is approximately 1.1. However, these factors may also vary by 10% in both directions. From the relationships above, and from Fig. 3, it can be deduced that the radii of curvature R_1, R_3, of the inward portions ~~3b, 4b~~ 4b, 3b of the blade are at least 50%, and preferably about 100%, greater than the radii of curvature R_2, R_4 of the radially outer portions ~~3a, 4a~~ 4a, 3a of back faces 4 and front faces 3 ~~front faces 3 and back faces 4~~, respectively. Further, the radius of curvature of radially inward portion 3b, 4b is no more than about four times the radius of curvature of the corresponding radially outer portion 3a, 4a. The radius of curvature of

the radially outer portion of the back face 4a is between about 5% and 50% greater than the radius of curvature of the radially outer portion of the front face 3a.